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"MULTIPLE SLIP EFFECTS ON NANOFLUID DISSIPATIVE FLOW IN A CONVERGING/ DIVERGING CHANNEL: A NUMERICAL STUDY"

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ABSTRACT

A mathematical model is developed for viscous slip flow and heat transfer in water/Ethylene containing metallic glycol-based nanofluids oxide nanoparticles, through а converging/diverging channel geometry. Our approach is based on the single-phase Tiwari-Das nanofluid model considering nanoparticles and base fluid masses as a substitute volume concentration of nanoparticles. The governing (dimensional partial differential) equations are transformed to a set of dimensionless ordinary differential equations with the help of similarity transformation, before being solved numerically using Maple17. Extensive validation of the velocity gradient and temperature solutions is achieved with the second order implicit finite difference Keller Box method (KBM). Further validation is included for the special case of noslip nanofluid flow in the absence of viscous heating. The effects of the emerging parameters namely velocity slip, thermal jump, channel apex angle, Eckert number, Prandtl number, Reynolds number and nano-particle volume fraction on velocity, temperature, skin friction and heat transfer rate are investigated in detail. Two different nanofluids are studied, namely water-Titanium oxide- and Ethylene glycol-Titanium oxide. Both convergent and divergent channels are addressed, and significantly different thermofluid characteristics are computed due to slip and viscous heating effects. The novelty of the current work is that it extends previous studies to include multiple slip effects and viscous heating (Eckert number effects) which are shown to exert a significant influence on heat and momentum transfer characteristics. The study is relevant to certain pharmaco-dynamics devices (drug delivery), next generation 3-D nanotechnological printers and also nano-cooling systems in energy engineering where laminar flows in diverging/converging channels arise.

Keywords: Convergent/divergent channel; nanofluids; viscous heating; momentum/thermal slip, nano-engineering devices.

1. INTRODUCTION

Heat transfer and viscous flows in non-uniform channels (diverging/converging) arise in many branches of applied physics, chemical and biomedical engineering including nozzle designs and protein exchange membrane fuel cells [1-4]. The classical theory of these so-called Jeffery-Hamel flows is lucidly documented in Schlichting [5] for Newtonian viscous fluids. Many extensions to the Newtonian case have been reported for a diverse array of complex fluids and with modified boundary conditions and body forces. Balmer and Kauzlaich [6] developed similarity solutions for steady non-Newtonian power law fluid flow in a converging or diverging two-dimensional channel with permeable walls, identifying a singularity caused by the fluid elasticity for the diverging geometry case at Deborah number of unity. Hooper et al. [7] studied analytically Jeffery-Hamel flow of a two-fluid system with variable viscosity observing that the velocity distribution becomes discontinuous with an increase in volume fluxbased Reynolds number. Hariharan et al. [8] used a Fourier series method to investigate peristaltic non-Newtonian flows of power law and Bingham fluids in a diverging tube with various wall wave forms (sinusoidal, multi-sinusoidal, triangular, trapezoidal and square), observing that divergence angle exerts a strong influence on occurrence of reflux near the tube wall even for zero fluxes and the thickness and evolution of the reflux region is markedly affected by the type of wave form. Wu et al. [9] conducted a theoretical and experimental study of effective planar micromixer flow featuring a meandering microchannel with convergingdiverging cross section, noting that fluid mixing in the microchannel is aided via Dean vortices and in particular the separation vortices generated by the converging-diverging cross section at large flow rate. Lamont et al. [10] measured heat transfer coefficient distributions in narrow diverging channels in gas turbine cooling systems. Stalio and Piller [11] used direct numerical simulation to computationally model unsteady forced convection in sinusoidal, symmetric wavy converging/diverging channels.

The above studies did not however consider *nanofluids*. Choi [12] first demonstrated that nanofluids possess enhanced thermophysical properties as compared to conventional heat transfer fluids (lower thermophysical properties). Nanofluids designate a solid-liquid mixture consisting of a fluid suspension having ultra-fine nanoparticles. The size of these suspended particles is typically of the order of few nano meters. The commonly used nanoparticles are Al₂O₃, CuO, TiO₂, ZnO and SiO₂. The significant features of nanofluids include higher thermal conductivity and viscosity as compared to the base fluid and stable nature of the suspension. The volume fraction of nanoparticles is normally engineered to be 3% to 5% so that the

nanofluid exhibit mechanical behavior similar to the base fluid. In recent years nanofluids have filtered into an incredibly diverse number of industries including petroleum [13], medical [14], chemical polymer coating [15], wound treatment [16], nano-materials processing [17, 18], microbial fuel cell enhancement [19], drug delivery (pharmaco-dynamics) [20] and even protein detection technologies [21]. Comprehensive reviews of heat transfer characteristics of nanofluids have been presented in [22-25]. Extensive efforts have been made to develop robust mathematical models for the thermal conductivity enhancement features of nanofluids. Suggested mechanisms for this include dispersion of nanoparticles, turbulence and microconvection, and also rotation of the nanoparticles. A popular model which is relatively simple to implement is that of Tiwari and Das [26] in which nanoparticle contribution is simulated via volume fraction rather than in a separate species conservation equation. His model has been employed to analyse a variety of metal-oxide nanofluids including zinc oxide, copper oxide, titanium oxide, aluminium oxide and others. Recent studies employing the Tiwari-Das formulation include Tripathi and Bég [27] who considered peristaltic propulsion of nanofluids in pharmacological systems. In the context of diverging channel flows, Akbar and Nadeem [28] used the Tiwari-Das approach to simulate peristaltic transport of viscoelastic nanofluids. Recently, various applications of the Tiwari-Das nanofluid model [26] have been analysed by a number of researchers [29-32].

In the present study we use **MAPLE17** numerical quadrature [33] to study the dissipative nanofluid flow and heat transfer in a converging/diverging channel with the Tiwari-Das model. Solutions are obtained for the velocity, temperature, wall shear stress and heat transfer rate (Nusselt number) distributions for a variety of the emerging parameters. Verification of the solutions for the general slip model is attained with the implicit finite difference Keller Box Method (**KBM**). Further validation for the no-slip case is conducted with the no-slip case reported by Moradi *et al.* [35] in the absence of viscous heating. The transformed ordinary differential boundary value problem, which is nonlinear, also provides a good benchmark for further more elaborate numerical simulations with for example computational fluid dynamics (CFD) codes [34] and significantly extends earlier investigations which have been confined to Jeffery-Hamel *non-slip flows without viscous heating effects* [35]. The present work extends the current research by including slip (momentum and thermal) and viscous heating effects for the first time in nanofluid Jeffrey-Hamel flows. It is relevant to cooling enhancement in energy systems and also biomedical devices (nozzle delivery) employing nanofluids where

diverging/converging channels feature in novel nozzle configurations, diffusers, inlet and outlet geometries.

2. MATHEMATICAL NANOFLUID MODEL

The flow model and coordinate system is displayed in **Fig. 1**, in a polar (r, θ) coordinate system. Nanofluid flow from a source/sink is studied at the apex of a converging/diverging channel (i.e. *non-parallel* solid walls intersecting) with intersection angle 2α .



Fig 1: Nanofluid Jeffery-Hammel flow geometry (diverging case)

Forced convection heat transfer takes place. Thermal dispersion and heat generation effects are neglected. With viscous dissipation present, which is important in incompressible flows, by means of Tiwari–Das nanofluid's model (Tiwari and Das [26]), the appropriate balance equations for two-dimensional Jeffery-Hamel nanofluid flow are the mass, momentum and heat conservation which are given below (Dinarvand and Rostami [29]).

$$\nabla \cdot \mathbf{v} = 0 \tag{1}$$

$$\rho \left[\frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot \nabla) \mathbf{v} \right] = -\nabla p + \mu \nabla^2 \mathbf{v}, \qquad (2)$$

$$\frac{\partial T}{\partial t} + \mathbf{v} \times \nabla \mathbf{T} = \alpha \nabla^2 T \tag{3}$$

Here **v** is velocity vector, *T* is temperature. $\nabla^2 = \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial}{\partial \theta^2}$. The conservation equations in polar coordinates in the absence of body forces and under aforementioned assumptions become:

Mass conservation:

$$\frac{\rho_{nf}}{r}\frac{\partial(ru)}{\partial r} = 0 \tag{4}$$

r-Momentum conservation:

$$u\frac{\partial u}{\partial r} = -\frac{1}{\rho_{nf}}\frac{\partial p}{\partial r} + \frac{\mu_{nf}}{\rho_{nf}}\left(\frac{\partial^2 u}{\partial r^2} + \frac{1}{r}\frac{\partial u}{\partial r} + \frac{1}{r^2}\frac{\partial^2 u}{\partial \theta^2} - \frac{u}{r^2}\right)$$
(5)

θ-Momentum conservation:

$$-\frac{1}{\rho_{nf}r}\frac{\partial p}{\partial \theta} + \frac{2\mu_{nf}}{\rho_{nf}r^2}\frac{\partial u}{\partial \theta} = 0$$
(6)

Heat (energy) conservation:

$$u\frac{\partial T}{\partial r} = \alpha_{nf} \left(\frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} + \frac{1}{r^2} \frac{\partial^2 T}{\partial \theta^2} \right) + \frac{\mu_{nf}}{(\rho C_p)_{nf}} \left(4 \left[\frac{\partial u}{\partial r} \right]^2 + \frac{1}{r^2} \left[\frac{\partial u}{\partial \theta} \right]^2 \right) = 0$$
(7)

The appropriate boundary conditions are specified as follows:

Symmetry (channel centre line):
$$\frac{\partial u}{\partial \theta} = 0; \frac{\partial T}{\partial \theta} = 0, u = U$$
 (8a)

Wall (-slip):
$$u = -N_1 v_{nf} \frac{\partial u}{\partial \theta}$$
; $T = T_w - N_1 \frac{\partial T}{\partial \theta}$ (8b)

Here *U* is an arbitrary velocity, *u* is radial velocity, *T* is temperature, *p* is pressure. The effective density ρ_{nf} , the effective dynamic viscosity μ_{nf} , the effective heat capacity $(\rho C_p)_{nf}$, effective thermal conductivity k_{nf} and thermal diffusivity α_{nf} of the nanofluid are defined following Tiwari and Das [26] as follows:

$$\rho_{nf} = (1 - \varphi) \rho_f + \varphi \rho_s, \quad \mu_{nf} = \frac{\mu_f}{(1 - \varphi)^{2.5}}, \quad (\rho C_p)_{nf} = (1 - \varphi) (\rho C_p)_f + \varphi (\rho C_p)_s$$

$$\frac{k_{nf}}{k_f} = \frac{k_s + 2k_f - 2\varphi (k_f - k_s)}{k_s + 2k_f + 2\varphi (k_f - k_s)},$$
(9a)

$$\alpha_{nf} = \frac{\kappa_{nf}}{(\rho C_p)_{nf}} \tag{9b}$$

Here ϕ is the solid volume fraction of the nanoparticles, ρ_f is the base fluid density, ρ_s is the nanoparticle (solid) density and μ_f is the base fluid dynamic viscosity. The nonlinear primitive boundary value problem defined by Eqns. (4)-(7) under boundary conditions (5a, b) generally required a numerical solution via explicit finite differences or a finite element method. It can however be significantly simplified by observing from the continuity equation that the radial velocity, *u*, must be of the form:

$$f = ur \tag{10}$$

We further introduce the following similarity variables:

$$F = \frac{f}{rU}; \ \eta = \frac{\theta}{\alpha}; \ \Theta = \frac{T}{T_w}$$
(11)

where *F* is a dimensionless radial velocity function, η is dimensionless angular coordinate and Θ is non-dimensional temperature. Elimination of the pressure, *p*, between Eqns. (5) and (6) leads to:

Momentum:

$$\frac{d^{3}F}{d\eta^{3}} + 2\alpha \operatorname{Re}[(1-\varphi)^{2.5}(1-\varphi+\varphi\frac{\rho_{s}}{\rho_{f}}]F\frac{dF}{d\eta} + 4\alpha^{2}\frac{dF}{d\eta} = 0$$
(12)

Energy:

$$\frac{1}{\left(1-\varphi+\varphi\frac{(\rho C_p)_s}{(\rho C_p)_f}\right)} \left[\frac{k_{nf}}{k_f}\frac{d^2\Theta}{d\eta^2} + \frac{\Pr Ec}{(1-\varphi)^{2.5}} \left[4\alpha^2 F^2 + \left(\frac{dF}{d\eta}\right)^2\right]\right] = 0$$
(13)

The transformed boundary conditions take the form:

At
$$\eta = 0$$
: $F(0) = 1$; $\frac{dF(0)}{d\eta} = 0$; $\frac{d\Theta(0)}{d\eta} = 0$ (14)

As
$$\eta \to 1: F(1) = -\frac{1}{(1-\phi)^{2.5}(1-\phi+\phi\frac{\rho_s}{\rho_f})} a F'(1); \ \Theta(1) = 1-b \Theta'(1)$$
 (15)

In Eqns. (12) and (13), the Reynolds number (Re), Eckert number (Ec) and Prandtl number (Pr) and velocity slip (a), thermal slip (b) are defined as:

$$\operatorname{Re} = \frac{U_{\max} r \rho_{f} \alpha}{\mu_{f}}, \operatorname{Ec} = \frac{U^{2}}{(c_{p})_{f} T_{w}}, \operatorname{Pr} = \frac{\mu_{f} (C_{p})_{f}}{k_{f}}, a = \frac{N_{1} v_{f}}{\alpha}, b = \frac{D_{1}}{\alpha}$$
(16)

For a *diverging* channel, in the Reynolds number $\alpha >0$, $U_{max}>0$; for the *converging* channel case, $\alpha <0$, $U_{max}<0$. In engineering simulations we are interested not only in the velocity and temperature functions, but also certain gradient functions of these variables. The nondimensional skin friction coefficient, C_f , may be defined in terms of transformed variables, thus:

$$C_{f} = \frac{\tau_{w}}{\rho_{nf} U_{\text{max}}^{2}} = \frac{1}{\text{Re}(1-\phi)^{2.5}} \frac{dF(1)}{d\eta}$$
(17)

Here the shear stress at the walls, τ_w , is:

$$\tau_{w} = \frac{\mu_{nf}}{r} \frac{du}{d\theta}$$
(18)

The dimensionless Nusselt number (wall heat transfer rate) is given by:

$$Nu = \frac{rq_w}{k_f T_w} \Big|_{\theta=\alpha} = -\frac{rk_{nf} \frac{dT}{dr}}{k_f T_w} \Big|_{\theta=\alpha}$$
(19)

In terms of the transformed variables we have:

$$Nu = -\frac{1}{\alpha} \frac{k_{nf}}{k_f} \frac{d\Theta(1)}{d\eta}$$
(20)

The fifth order nonlinear boundary value problem described in Eqns. (12)-(13) under boundary conditions (14), (15) is unlikely to yield analytical solutions. We therefore develop numerical solutions using **MAPLE17**. The methodology is described in detail in [36-38]. Several important special cases of the present flow model may be retrieved. The flow model describes Newtonian Jeffery-Hamel flow as $\varphi \rightarrow 0$ (nanoscale effects vanish). When Ec = 0 viscous

heating is negated. When both nanoscale and viscous heating effects are omitted the model reduces to that studied in [35]. In the general model, a homogenous distribution of nanoparticles is assumed in the nanofluid i.e. a *dilute* suspension. The nanofluid consists of *two-component* mixtures (nano-solid-particles and the base fluid are in thermal equilibrium) and no slip occurs between them. Slip arises *only* at the channel walls.

3. MAPLE SOLUTIONS

The boundary value problem (BVP) is solved using MAPLE 17 numerical integration quadrature based on Runge-Kutta Fehlberg shooting algorithms. This is a highly efficient methodology for BVPs with infinity boundary conditions. A Runge-Kutta-Fehlberg fourthfifth order numerical algorithm (RKF45) is employed, which is readily available in the symbolic computer software MAPLE17. This method has been employed extensively in fluid mechanics simulations, including nanofluid dynamics e.g. radiative slip flow of magnetic nanofluids [37], magnetic nanofluid flow from a stretching surface [38], power-law nanofluid flow in porous media [39]. Other applications include Marangoni (surface-tension) driven nanofluid transport in biopolymers [40] and non-Newtonian petrochemical mass transfer in porous media [41]. The RK45 algorithm is based on a collocation method in which a finitedimensional space of candidate solutions is selected (usually, polynomials up to a certain degree) and a number of points in the domain (called *collocation points*), and a solution selected which satisfies the given equation at the collocation points. The details of **RFK45** methods can be found in the ref. [37, 38]. MAPLE is an excellent symbolic software with many libraries of built in ready-to-use numerical solvers for ordinary and partial differential problems. In MAPLE the RK45 quadrature is used to yield temperature and stream function. The appropriate velocity is then computed in a sub-iteration loop. The robustness and stability of this numerical method is therefore well established- it is highly adaptive since it adjusts the quantity and location of grid points during iteration and thereby constrains the local error within acceptable specified bounds. Many different wall boundary conditions are easily accommodated. The stepping formulae although designed for nonlinear problems, are even more efficient for any order of linear differential equation and are summarized below [37]:

$$k_0 = f\left(x_i, y_i\right),\tag{21}$$

$$k_1 = f\left(x_i + \frac{1}{4}h, y_i + \frac{1}{4}hk_0\right),$$
(22)

$$k_{2} = f\left(x_{i} + \frac{3}{8}h, y_{i} + \left(\frac{3}{32}k_{0} + \frac{9}{32}k_{1}\right)h\right),$$
(23)

$$k_{3} = f\left(x_{i} + \frac{12}{13}h, y_{i} + \left(\frac{1932}{2197}k_{0} - \frac{7200}{2197}k_{1} + \frac{7296}{2197}k_{2}\right)h\right),$$
(24)

$$k_4 = f\left(x_i + h, y_i + \left(\frac{439}{216}k_0 - 8k_1 + \frac{3860}{513}k_2 - \frac{845}{4104}k_3\right)h\right),\tag{25}$$

$$k_{5} = f\left(x_{i} + \frac{1}{2}h, y_{i} + \left(-\frac{8}{27}k_{0} + 2k_{1} - \frac{3544}{2565}k_{2} + \frac{1859}{4101}k_{3} - \frac{11}{40}k_{4}\right)h\right),$$
(26)

$$y_{i+1} = y_i + \left(\frac{25}{216}k_0 + \frac{1408}{2565}k_2 + \frac{2197}{4101}k_3 - \frac{1}{5}k_4\right)h,$$
(27)

$$z_{i+1} = z_i + \left(\frac{16}{135}k_0 + \frac{6656}{12825}k_2 + \frac{28561}{56430}k_3 - \frac{9}{50}k_4 + \frac{2}{55}k_5\right)h.$$
 (28)

Here y denotes fourth order Runge-Kutta phase and z is the fifth order Runge-Kutta phase. An estimate of the error is achieved by subtracting the two values obtained. If the error exceeds a specified threshold, the results can be re-calculated using a smaller step size. The approach to estimating the new step size is shown below:

$$h_{new} = h_{old} \left(\frac{\varepsilon h_{old}}{2|z_{i+1} - y_{i+1}|} \right)^{\frac{1}{4}}.$$
(29)

The **MAPLE17** code has been extensively validated for many nonlinear problems [37-41] and confidence in the current computations is very high. In order to analyse the sensitivity of the physical model to different thermophysical parameters, extensive computations have been conducted and are documented in **Figs. 2-11**. In each case the following nanofluid properties must be determined in eqns. (9), (10) i.e. $\frac{\rho_s}{\rho_f}$ (density ratio in eqn (9)) and $\frac{(\rho C_p)_s}{(\rho C_p)_f}$ (i.e. specific

heat capacity ratio) and $\frac{k_{nf}}{k_f}$ (i.e. thermal conductivity ratio) in eqn. (10). We examine a number of nanofluids e. g. water-Titanium oxide- and Ethylene glycol - Titanium oxide, as documented

in [26]. Further graphs can be generated for Aluminium oxide and copper oxide, using

properties [27]. Velocity gradient, temperature, skin friction and Nusselt number are all computed.

4. VALIDATION OF GENERAL MODEL WITH KELLER BOX METHOD (KBM)

In order to justify our results, the Keller-Box implicit difference method (**KBM**) is also used to solve the same equations. This second order accurate method is ideal for *parabolic* problems e.g. boundary layer flows, although it can be used for fully developed channel flows also. Recently the **KBM** algorithm has successfully resolved a number of nonlinear magnetohydrodynamics and nanofluid dynamics problems including micropolar nanofluid enrobing flows [42], Hall MHD generator transport [43], viscoelastic flows in porous media [44] and biological micro-organism propulsion [45]. The detail of the KBM is elaborated by Keller [46]. The numerical algorithm is performed in **MATLAB** on an Octane SGI workstation and computes in seconds. The method demonstrates excellent stability, convergence and consistency, as elaborated by Keller [47]. In KBM, there are 4 key steps:

(I)Reduction of Eqns. (12) - (13) into a system of first order ordinary differential equations. Thus, the coupled differential equations of third order in F (η) and second order in Θ (η) are reduced to a system of five simultaneous equations of first order for five unknowns. Boundary conditions (14 and (15) are also transformed.

(II)In the second step derivatives are approximated in the new system of first order equations with central difference approximations by considering a net rectangle in the $x\eta - plane$ and the net points are defined as $\eta_0 = 0$, $\eta_j = \eta_{j-1} + h_j$ n = 1, 2, 3...J; j = 1, 2, 3...J and $\eta_j = \eta_{\infty}$. Here h_j is the $\Delta \eta$ -spacing and n, j are just the sequences of numbers that indicate the coordinate location. The centering midpoint $\left(\eta_{j-\frac{1}{2}}\right)$ of the segment is obtained by using the following finite difference approximations.

(III) In the third step the emerging nonlinear algebraic equations are linearized with Newton's method by using iterates of the form $\binom{i+1}{j} = \binom{i}{j} + \delta\binom{i}{j}$ and then cast into matrix vector form. (IV) Finally, in the fourth step, the linearized algebraic equations are solved using a block tridiagonal elimination scheme implemented in MATLAB software with the suitable initial solution. This method is unconditionally stable has a second order accuracy and is relatively easy to program, thus making it highly attractive for engineering analysis. For this iterative scheme to solve the system of equations, a convergence criterion is required. This is specified

as follows: when the difference between two successive approximations is sufficiently small ($\leq 10^{-5}$) the solutions are taken to have converged to the requisite accuracy.

Comparison computations (which relate to the appropriate **MAPLE 17** green dotted lines only) are denoted by the *blue triangles* in **Figs 2a, 3a, 4a, 5a** and **6a**. In each graph in other words we have verified one **MAPLE17** solution and excellent agreement is achieved. Confidence in **MAPLE17** shooting solutions is therefore justifiably high.

5. VALIDATION WITH NO-SLIP CASE AND ABSENCE OF VISCOUS HEATING

Due to unavailability of experimental data, it is not possible to compare with the experimental data. However, we have included a comparison table with the published data of a simpler previous study [35]. Tables 1 and 2 are comparisons of the MAPLE 17 code for the special reduced no-slip (N₁ =0) case and for zero volume fraction ($\phi = 0$) with [35]. At all Reynolds numbers and vertex angles of the diverging/converging channel, excellent correlation is achieved for the stream function $f(\eta)$ and skin friction f''(0) with MAPLE RK45 and the solutions of Moradi *et al.* [35]. Confidence in the MAPLE RK45 code is therefore justifiably very high.

	Moradi et	al. [35] numeric	al results	Present numerical		results
	Re = 110	Re = 80	Re = 50	Re = 110	Re = 80	Re = 50
η	$\alpha = 3^{\circ}$	$\alpha = -5^{\circ}$	$\alpha = 5^{\circ}$	$\alpha = 3^{\circ}$	$\alpha = -5^{\circ}$	$\alpha = 5^{\circ}$
0	1	1	1	1	1	1
0.1	0.979236	0.995961	0.982431	0.97923571	0.99596063	0.98243124
0.2	0.919266	0.983276	0.931226	0.91926589	0.98327554	0.93122597
0.3	0.826534	0.96018	0.850611	0.82653362	0.96017991	0.85061063
0.4	0.710221	0.923522	0.746792	0.71022119	0.92352159	0.74679081
0.5	0.580499	0.868459	0.626849	0.58049946	0.86845888	0.62694818
0.6	0.446935	0.788091	0.498234	0.44693507	0.78809092	0.49823446
0.7	0.317408	0.673144	0.366966	0.31740843	0.67314363	0.36696635
0.8	0.197641	0.511991	0.238124	0.19764109	0.51199109	0.23812375
0.9	0.09123	0.291559	0.115152	0.09123042	0.29155874	0.11515193
1	0	0	0	0	0	0

Table 1: Numerical values of the function $f(\eta)$ for Newtonian fluid ($\phi = 0$)

$\alpha = 0.1$	Moradi e	et al. [35]	Present results		
	TiO ₂	Cu	TiO ₂	Cu	
Re = 10	-0.236316	-0.228015	-0.23631589	-0.22801492	
Re = 30	-0.0630751	-0.0547215	-0.063075067	-0.05472165	
Re = 50	-0.0284339	-0.023245	-0.028433869	-0.02324493	
$\alpha = 0.2$					
Re = 10	-0.319488	-0.302882	-0.31948789	-0.30288148	
Re = 30	-0.0869483	-0.070249	-0.086948288	-0.07024876	
Re = 50	-0.0404258	-0.0241233	-0.040425766	-0.024123292	

Table 2: Numerical values of skin friction for Newtonian fluid ($\phi = 0$)

6. MAPLE17 COMPUTATIONAL RESULTS AND DISCUSSION

Figs 2a,b illustrate the evolution of non-dimensional velocity gradient for water-TiO₂ and ethylene glycol- TiO₂ nanofluids, with nanoparticle volume fraction (ϕ) and momentum slip parameter (*a*). The evolution of velocity has already been elaborated in other studies e.g. Moradi *et al.* [35] and is well-known. We therefore elect here to examine velocity gradient response instead. In all figures the viscous heating is strong (*Ec* = 5), the channel is *diverging* ($\alpha = 3^{\circ}$), flow is *laminar* (*Re* =50) and the *thermal slip* is weak (*b* = 0.5). The range $0 \le \eta \le 1$ corresponds to the channel semi-space i.e. from the longitudinal axis to the upper wall. Since reflective symmetry is achieved via the boundary conditions and geometry it is not necessary to plot the lower semi-space.



Fig. 2: Effects of volume fraction of nanoparticles and slip parameter on dimensionless velocity gradient for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids. [Blue triangles represent the appropriate KBM solution validating the green dotted line MAPLE17 case].



Fig. 3: Effects of *converging* channel angle and Reynolds number on dimensionless velocity gradient for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids. [Blue triangles represent the appropriate KBM solution validating the green dotted line MAPLE17 case].



Fig. 4: Effects of *diverging* channel angle and Reynolds number on dimensionless velocity gradient for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids. [Blue triangles represent the appropriate KBM solution validating the green dotted line MAPLE17 case].



Fig. 5: Effects of volume fraction of nanoparticles and thermal slip parameter on dimensionless temperature for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids. [Blue triangles represent the appropriate KBM solution validating the green dotted line MAPLE17 case].



Fig. 6: Effects of converging channel angle and Reynolds number on dimensionless temperature for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids. [Blue triangles represent the appropriate KBM solution validating the green dotted line MAPLE17 case].



Fig. 7: Effects of diverging channel angle and Reynolds number on dimensionless temperature for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids.



Fig. 8: Skin friction as a function of velocity slip, volume fraction of nanoparticles and Reynolds number for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids (converging case).



Fig. 9: Skin friction as a function of velocity slip and converging-diverging channel angles for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids.



Fig. 10: Nusselt numbers as function of velocity slip, volume fraction of nanoparticles and Reynolds number for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids.



Fig. 11: Nusselt numbers as a function of Eckert number and converging-diverging channel angles for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids.

Radial velocity gradient (which is prescribed zero magnitude at the channel centre line) grows steadily from the centre line ($\eta = 0$) towards the channel wall; however with greater momentum slip, it peaks in close proximity to the wall and not at the wall, whereas in the absence of slip it attains a maximum at the wall. With greater momentum slip for both the both water-TiO₂ and ethylene glycol-TiO₂ nanofluids [46], velocity gradient is depressed indicating that the nanofluid shears slower and decelerates. With increasing nanoparticle volume and momentum slip present (a= 0.5, 1) the velocity gradient is reduced for both nanofluids. However in the no-slip case (a = 0) this behaviour although present for most of the channel half space, is reversed near the wall where volume fraction increasing is observed to elevate velocity gradient. Generally therefore flow deceleration accompanies greater volume fractions with velocity slip present whereas the converse behaviour is observed in the no-slip scenario.

Figs 3a,b present the radial velocity gradient distributions with varying Reynolds number (*Re*) and channel semi-vertex angle (α). Only converging channel cases are examined ($\alpha > 0$). Generally for the most part of the semi-channel space, increasing Reynolds number serves to accelerate the flow and increases velocity gradient magnitudes. With increasing semi-vertex

angle, acceleration is also caused in this large extent of the channel semi-space. However closer to the wall (boundary), the reverse effect is computed. The nanofluid flow is markedly decelerated, although not to the same degree as the acceleration in the zone far from the wall. The inertial body force associated with larger Reynolds numbers is therefore counter-productive in the near-wall zone whereas it is beneficial in the near centre-line zone. The implication is that enhanced momentum transfer is achieved closer to the symmetry axis of the channel rather than near the boundary, with greater inertial effects. With wider vertex angles of the channel, the velocity is also boosted near the channel centre and around this zone as compared with the boundary (wall) zone where greater drag is induced. Very little tangible difference is observed between the water-TiO₂ (Fig. 3a) and ethylene glycol-TiO₂ nanofluids (Fig. 3b), despite the difference in viscosity of the base fluids.

Figs. 4a, b present the evolution of velocity gradient with semi-vertex angle (divergent, $\alpha < 0$) and Reynolds number (Re) with fixed momentum and thermal slip and weak viscous heating at nanoparticle volume fraction of 0.2. An increase in divergent semi-angle generally retards the flow from the centre line for the majority of the space until the near-wall zone where it induces a *weak acceleration*. The response is therefore opposite to that of increasing convergent angle (Figs. 3a,b). With greater divergence of the vertex, momentum distribution is adversely affected. The flow acceleration achieved with narrower angles is lost and the nanofluid flow behaves more like a diffuser system. This configuration is therefore better for more controlled deployment of fluids in medical devices e.g. nasal sprays [47], where more homogenous distribution of nanoparticles may be achieved in the core flow region. However convergent angles (Figs. 3a,b) are more suited to targeted, fast execution of drugs e.g. surface skin treatment (semi-powder-jet systems), divergent sections of de Laval nozzles in diabetic pumps [48] etc. Both geometric scenarios may therefore be exploited in different clinical applications. Figs. 4a and b, further demonstrate the principally decelerating influence of increasing Reynolds numbers. With greater inertial force (relative to viscous force), significant reduction in the flow is achieved for divergent channels, again testifying to the destruction of momentum in this type of geometry. These effects are computed for both water-TiO₂ (Fig. 4a) and ethylene glycol-TiO₂ nanofluids (fig. 4b), although slightly greater magnitudes of radial velocity gradient are attained in the former, in particular near and at the channel wall.

Figs. 5a, b illustrate the temperature distributions for various nano-particle volume fraction (ϕ) and thermal slip parameters (*b*) with fixed momentum slip and strong viscous heating at

Reynolds number of 40 and for a convergent channel. A substantial increase in temperature accompanies greater thermal slip values, for both water-TiO₂ (Fig. 5a) and ethylene glycol-TiO₂ nanofluids (fig. 5b). Thermal diffusion is therefore assisted with greater thermal jump at the channel wall. The nanofluid is significantly heated, although temperature is observed to drop from the centreline to the wall for any scenario. With greater nanoparticle volume fraction there is however a slight decrease in temperatures for water-TiO₂ (Fig. 5a) whereas there is more prominent reduction in temperature for ethylene glycol-TiO₂ nanofluid (fig. 5b). The general thermal enhancement properties of nanofluids, as predicted by many researchers, e.g. [24-26, 32], is therefore *not* computed for convergent channels.

Figs. 6a, b illustrate the temperature distributions for various semi-vertex angle (convergent, $\alpha > 0$) and Reynolds number (*Re*) with fixed momentum and thermal slip (a = b = 0.5), viscous heating (Ec = 5) and volume fraction ($\phi = 0.2$). Significantly higher temperatures are achieved with ethylene glycol-TiO₂ nanofluid (fig. 6b), than with water-TiO₂ (Fig. 6a) nanofluid. With greater semi-vertex angle, temperatures are strongly enhanced for both nanofluids. A wider channel convergent vertex therefore heats the nanofluid considerably compared with a narrower one. With greater Reynolds number, again a marked elevation in temperatures as achieved. A similar trend was reported also by Moradi *et al.* [35]

Figs. 7a,b present the influence of various semi-vertex angle (divergent, $\alpha < 0$) and Reynolds number (*Re*) again with fixed momentum and thermal slip (a = b = 0.5), viscous heating (*Ec* = 5) and volume fraction ($\phi = 0.2$). Converse to convergent channels, a substantial decrease in temperatures is caused with greater Reynolds numbers. However an increase in temperature does correspond to increasing divergent semi-vertex angle. Therefore both nanofluids are heated throughout the channel space irrespective of whether the vertex is converging or diverging. As with converging channels, however, greater temperatures are attained with ethylene glycol-TiO₂ nanofluid (fig. 7b) than with water-TiO₂ (Fig. 7a) nanofluid.

Figs 8a,b depict the skin friction plots for various thermophysical parameters. With greater nano-particle volume fraction (ϕ) a marked boost is achieved in skin friction, for both nanofluids, indicating considerable flow acceleration. Conversely with a small increase in Reynolds number, the flow is decelerated. This concurs with the trend in Fig. 3a where velocity gradients were noted to be lowered at the channel wall with greater Reynolds numbers. Approximately the same response is observed for both nanofluids i.e. the base fluid exerts very

little effect on the flow performance – the primary influence is from the nanoparticles, which as they are both Titanium oxide in figs 8a,b, do not alter profiles significantly. However the concentration of nano-particles simulated via the volume fraction clearly does modify the flow at the wall. Increasing momentum slip is also found to reduce skin friction.

Figs. 9a, b present the effects of velocity slip (*a*) and converging or diverging channel angles (α) for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids, on skin friction (*C*_f). With increasing momentum (velocity) slip skin friction is markedly depressed for both nanofluids. With increasing converging angle, skin friction is also decreased for both nanofluids, whereas with increasing diverging angle it is strongly increased.

Figs 10a, b illustrate Nusselt number (wall heat transfer rate) for various nano-particle volume fraction (ϕ), velocity slip (*a*) and Reynolds numbers (Re), again for a converging channel. Significant decay in Nusselt numbers accompanies increasing momentum slip values for both nanofluids. Generally Nusselt number is also enhanced with increasing Reynolds number and the effect is more pronounced as velocity slip increases. A strong elevation in Nusselt number is also observed with higher nanoparticle volume fraction. Higher concentrations of nanoparticles therefore increase the transfer of heat from the nanofluid to the channel wall causing a drop in nanofluid temperature in the channel i.e. a cooling effect is induced. Distinct from the skin friction plots (Figs 9a,b), massively greater magnitudes of Nusselt number are achieved with EG-TiO₂ nanofluids (Fig 10b) compared with Water-TiO₂ nanofluids (Fig 10a) confirming that the base fluid has a prominent role in thermal enhancement of the nanofluid.

Figs 11a,b illustrate the influence of Eckert number (*Ec*), velocity slip (*a*) and convergingdiverging channel angles (α) for (a) Water-TiO₂ and (b) EG-TiO₂ nanofluids. Again we note the enormously higher magnitudes of Nusselt number computed for EG-TiO₂ nanofluid in comparison with Water-TiO₂ nanofluid. With greater Eckert number, Nusselt number clearly demonstrates a *linear descent for diverging channels*, whereas it follows a *linear ascent for converging channels*. Heat transfer rate to the wall from the fluid is therefore greater for converging channels with higher Eckert number. This parameter which arises only in the heat conservation equation (13) embodies the relative significance of kinetic energy in the flow to enthalpy difference, as elaborated by Schlichting [1]. *Ec* >0 implies conversion of mechanical energy which is dissipated as heat in the nanofluid. This will heat the fluid and lead to greater heat transfer from the fluid to the channel wall, as observed in the red solid lines in both Figs 11a,b. With increasing converging angle the rate of ascent of the Nusselt number profiles is increased, whereas with increasing diverging angle it is decreased. Heat transfer rate from the nanofluid to the wall therefore is boosted with wider convergent channels whereas it is depressed with wide divergent channels. The opposite effect will be induced in diverging channels, as testified to by the dotted green lines in both figs 11a, b.

7. CONCLUSIONS

A new transport model has been presented for thermal and momentum slip effects on dissipative nanofluid heat transfer and flow in a diverging/converging 2-D channel with solid walls. The Tiwari-Das nanofluid formulation has been implemented. The transformed boundary value problem is solved by MAPLE 17. Extensive validation has been achieved with the finite difference implicit Keller Box Method (KBM). The influence of nanoparticle volume fraction, Reynolds number, Prandtl number, channel semi-apex angle and Eckert (viscous dissipation) number are addressed. Properties of different nanofluids i.e. density ratio, thermal conductivity ratio and specific heat capacity ratio are evaluated using Tiwari-Das data for different nanoparticles (Titanium oxide, copper, alumina). Two nanofluids are considered- Water-TiO₂ and (b) Ethylene Glycol (EG)-TiO₂ nanofluid. The present computations have shown that with increasing velocity slip, for both water-TiO₂ and ethylene glycol-TiO₂ nanofluids, the channel bulk flow is decelerated. Similarly with greater solid (nanoparticle) volume and in the presence of momentum slip, the flow is also retarded. With increasing semi-vertex angle, the channel flow is generally accelerated although at the wall, the contrary behaviour is computed. An increase in divergent semi-angle on the other hand is found to generally decelerate the flow from the centre line for the core flow region, whereas near and at the channel wall it results in a weak acceleration. Higher temperatures are achieved with greater thermal slip values, for both water-TiO₂ and ethylene glycol-TiO₂ nanofluids, whereas for greater nanoparticle volume fraction, temperatures are weakly decreased for water-TiO₂ (Fig. 5a) whereas a more significant decreases is observed for ethylene glycol-TiO₂ nanofluid. With greater diverging channel angle, a substantial decrease in temperatures is caused with greater Reynolds numbers, and the reverse effect is computed for converging channels. The present model has neglected non-Newtonian properties [49] of the nanofluid and also transient effects [50, 51]- these will constitute future studies.

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